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Bulletin December 1978.

#OPTIMIZATION OF SOLAR COLLECTOR SIZE TO

ACHIEVE THE HEAT REQUIREMENT ALONG

CERTAIN PERIOD"

BY

Mohamed Helmy El-Maghraby⁴ (B.Sc., M.Sc., Ph.D., IEEE member)

ABSTRACT

This paper presents a complete analysis and a numerical application for deducing the optimum solar collector size to satisfy the heat requirement per year for certain estimated life-time of the heating system. Total cost of the heating systems (collector and auxiliary one) is computed for various collector areas. The collector area corresponding to the minimum total cost yields the optimum collector size. This goal is very significant since with increasing collector size. the solar contribution will be greater but certain heat collected will wasted if there is no simulataneous demand. On other hand, it is unequivocal that a small collector of only a few square metres would be utilized all year yound. Various collector cost functions are assumed to establish its influence on both optimum and maximum collector size beyond which the estimated life-time cost would be more than that a conventional one.

Annual net solar contributions for different collector areas are computed. The behaviour of utilization rate and figure of merit for various areas of collector is explained and researched.

Eventually, this article reveals the effect of the capital cost of the collector size on the optimum and maximum collector areas.

1. INTRODUCTION

The purpose of attaining the optimum collector size is very significant since it achieves both the technical (heat requirement through certain period, say, year) and the economical (minimum costs of the heating system) constraints. The heating system includes solar collector and an auxiliary heating system.

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Much of radiant energy is absorbed when it falls on a matt black surface. This absorption depends on the type of absorber material. This complex phenomenon includes scattering, photon absorption, acceleration of electrons, multiple collisions, but the final influence is that this radiant energy of all wavelengths is degraded to heat. The temperature increases since the molecules of the surface will be excited. The absorption coefficient of the several types of black absorbers varies from 0.8 to 0.98. By the effect of conduction, some of this molecular movement (i.e. heat) is transmitted to other parts of the body, and some of it is re-emitted to the environment by convective and radiant processes.

The differece in temperature between the surface and the environment affects this emission of heat. The equilibrium temperature is reached when the rate of radiant heat input is equalled by the heat loss. If the surface of the absorber plate is covered by a sheet of glass with certain air space (20 --- 30mm) the heat loss is considerocbly reduced without much decreasing in the heat input. This is because of the selective transmittance of the glass. For short wave, it is highly transparent i.e. high temperature solar radiant, however, it is opaque for longer wavelength infra-red radiation emitted by the absorber plate below 100 °C. The glass causes some decrease of the radiation intensity on the absorber plate. This means that there is an optical loss in transmission. However, this is much less than the resultant saving on heat loss. The proportion transmitted is expressed by the transmission coefficient which has a constant value for diffuse radiation, although for direct radiation, it is a function of the angle of incidence.

Some 25% of all energy consumed is used for the heating of buildings and domestic hot water. The lowest grade of energy is required for the space and water heating. With low temperature collection, the highest collection efficiencies are obtainable.

A tracking mechanism is required for the focusing devices and respond to direct radiation only. Flat plate collectors can utilize both diffuse and direct radiation and may be fixed in one particular position. They replaces an enclosing element such as a wall or a roof i.e. they may become part of the building envelope.

2. OPTIMIZATION OF THE SOLAR COLLECTOR SIZE WITH THE REQUIRED

CONSTRAINTS:

2.1. Flat Plate Collectors

If a certain thermal fluid (e.g. air or water) is circulated as a carrying medium in thermal contact with the E. 64. Mansoura Bulletin December 1978.

absorber plate, then it will be heated and thus some of the heat absorbed by the plate will be removed. Then, the temperature of the plate is roduced to below the equilibrium temperature and this will decrease the heat loss.

The solar collector plate itself can be any metal sheet, incorporating water channels. It may be-in its simplest casean ordinary central heating radiator panel. Many steel, copper an aluminium products are on the market which may be suitable.

Some form of header or manifold connects the water channels at top and bottom. The header should have a cross sectional area larger than the aggregate area of the channels served to ensure a balanced and uniform flow in all channels.

Surface finish of the absorber plate may be a matt black raint such as a chalk-bourd. black, with an appropriate rust inhibiting primer.

The so-called "selective surfaces have a high absorption and emission coefficient for the 200-2000 nm solar radiation, but a much lower a and e value for the longer infra-red (up to 20000 nm) emitted by bodies at a temperature below 100°C. The a/e ratio is a measure of their performance i.e. the ratio of absorption coefficient for solar radiation to the emission coefficient at operating temperatures.

2.2. Annual Heat Requirement

The annual heat requirement of a certain building will depend on the following two factors:

a) The climatic parameters

The climate can be characterized by the degree-day concept. This can be expressed as the annual cumulative temperature deficit which is the sum of the products of temperature differences and their duration.

A reference level is taken as indoor temperature (t_{0}) (say 18°C). For every day, the mean outdoor temperature (t_{0}) is established and then the temperature difference is taken as $(t_{1} - t_{0})$. If, for example, the mean (t_{0}) is 2°C for three days, 3 x (18 - 2) = 48 degree-days are added to the sum. This is calculated for all days of the year, whenever the t_{0} is less than the reference level. If these values are multiplied by 24, we have the required climatic parameter in a more convenient form: the number of degree-hours (degc.h.)

b) The building parameter

The building will be characterized by the specific heatloss rate concept. This rate is the total heat loss rate per unit temperature different. The total heat-loss is the sum of two components: 1- The heat loss rate through the building envelope (Q_{n}) can be expressed as $Q_{c} = (\sum A \times U) \Delta t$ where A = area of each element (m^2) U = thermal transmittance of each element(W/m² degc) $\Delta t = temperature difference (degc).$ and 2- The ventilation heat loss (Q_) and is given by Q. = 0.36 x V x N x _t where V = volume of space (m³)N = Number of air changes/h $\Delta t = temp. difference (degc).$ $Q(Total heat loss) = Q_c + Q_v$ $= ((ZA \times U) + 0.36 \times V \times N) \Delta t$ The specific heat loss rate = $\frac{Q}{\Delta t}$ = (ZA x U) + 0.36 x V x N Dimensionally Ventilation rate in $\frac{m^2}{h}$ $\frac{W}{\text{degc}} = \underline{m}^2 \frac{W}{\underline{m}^2 \text{ degc}} + \frac{Wh}{\underline{m}^3 \text{ degc}} \cdot \frac{\underline{m}^3}{h}$

For a small well insulated house, the value of this may be as low as 200 W/degc and as much as 1000 W/degc for a large loosely planned residence.

The annual heating requirement is the product of the two parameters: the degree-hours and the specific heat loss rate

If a combined space and water system is considered, we add the water heating requirement to the above value. The water heating requirement will be the product of the daily hot water consumption, 365, the specific heat of water (1.16 Wh/litre x degc) and the increase from cold supply temperature to the required hot water temperature. E.66. Mansoura Bulletin December 1978

2.3. Solar contribution, figure of merit, and utilization rate.

The supply of the solar heat is out of phase with the heating demand. Much of the energy available in the summer will be wasted, as there is no simultaneous demand. However, the collection system will be self-regulating to some extent: i.e. there is no further collection when the water temperature reaches the stability point i.e. the heat loss from the collector will equal the solar gain.

The following seven parameters influence the amount of energy usefully collected: incident energy, optical loss through transparent cover, absorption properties of the receiving surface, heat transfer properties of the absorber from surface to fluid i.e. the plate effeciency, thermal transmittance of transparent cover, which is a factor of heat loss, collection temperature which depends on: the fluid flow rate and the fluid temperature at entry to collector, and the external air temperatures.

The collection efficiency (ratio of the utilized energy to incident energy) will be affected by these parameters. It may be as high as 70% but it can be as low as 30%.

It is suitable to assume an average collection efficiency of 30% for the heating season with collection temperature about $40 \rightarrow 50^{\circ}$ C for the purposes of a crude estimate.

The intensity of radiation uniformly · ies, however, the cumulative total for a day or even for a month will provide a sufficient basis for an estimate.

The average monthly totals are measured on a horizontal plane, and totals for a vertical south facing wall are calculated in addition for an optimally tilted plane for the site under consideration.

If we tabulate these values of radiation for certain site, we can easily remark that, there is a certain constant ratio on average between the energy received by the optimally tilted plane and that of the horizontal one. Then if the values for a horizontal plane are multiplied by the product of this ratio and the assumed 30% efficiency which would yield the amount of collected energy for the period taken (say, one month). Then, total collection energy is computed for each month by multiplying the preceding unit collection energy (in KWh/m²) by the solar collector area in m².

The effective solar contribution is given by the total collection energy in KWh but not more than the space heat requirement in KWh.

The highest collection efficiency does not necessarily mean the most economic system since this high efficiency can often only be achieved by a very expensive installation. Then a balance must be found between capital expenditure and resultant savings in running costs. Figure of merit (FM) is a useful expression of this costeffectiveness of an installation.

$FM = \frac{Value \text{ of energy saved by the install. in certain period}}{Extra cost of installation over a convent. one}$

The system will be competitive if the figure of merit reaches a value of 1- in certain proposal-i.e. if, say, 10 years energy saving will equal the capital cost. FM, simply, gives a convenient comparative value. The energy saved is determined by establishing first the annual heat requirement, the amount of energy actually contributed by the solar collector and eventually this must be given a monetary value per KWh which depends on the type of fuel. FM is, then given by dividing the monetary value of the energy saved (or solar collected energy) in the periods assumed (5, 10 and 15 years) by the extra cost of the solar heating system which is given by the product of the collector cost (f/m^2) by the collector area in m^2 .

The utilization rate is assigned by the ratio of the cumulative net solar contribution along the year and the total collected energy by the collector considered. For certain heat demand during a period of, say, one year, the utilization rate decreases as the solar collector area increases.

2.4. Numerical application

This application reveals that an important amount of the heat collected will be wasted as there is no simultaneous demand. It is clear that a small collector of only a few square metres would work all year round; there would be a demand at all times for the heat it produces (It is used for water heating as well as space heating) we say, then, that its utilization rate would be one. Its total contribution or its portion of the total heat demand would be small. By increasing the collector size, the contribution will be greater and the utilization rate would be reduced.

The deduction of the optimum collector size can be established by the minimization of the total cost of the expected lifetime of the heating system.

The following features are taken into consideration:

- 1) The collection cost is assumed to have linear function with different slopes (£ $10/m^2$ and £ $15/m^2$) to derive the sensitivity of the optimum collector size to these variations.
- 2) The expected life-time of the auxiliary heating system is assumed to have various years (5, 10 and 15) to get its influence on the optimum collector size.

- 3) Several types of fuel used in the auxiliary heating system are taken to investigate what type that yields minimum total cost.
- 4) The problem is researched for different values of the specific heat loss rate of the house (0.2 KW/degc -> 1.0KW/degc). The values taken are 0.25, 0.50, and 1.0 KW/°C.
- 5) The annual set solar contribution with various areas of collector is calculated.
- 6) It is assumed that the optimally tilted plane receives about 1.5 times as much energy as the horizontal surface.

Table (1) yields the average monthly cumu] tive totals of the radiation of certain site measured on a horizontal plane and the calculated totals for a vertical south facing wall as well as for an optimally tilted (34°C) plane at the country chosen.

| Month | Horizontal (KWh/m ²) | South vertical KWh/m ² | South 34° tilt KWh/m ² |
|-------|-------------------------------------|--------------------------------------|--------------------------------------|
| Jan | 18.3 | 30.3 | 29.4 |
| Feb. | 30.9 | 47.3 | 51.6 |
| Mar. | 60.6 | 61.8 | 81.8 |
| Apr. | 111.9 | 75.9 | 137•1 |
| May | 123.2 | 57 •2 | 133.2 |
| Jun. | 150.4 | 53.8 | 155•7 |
| Jul. | 140.4 | 53.6 | 142.1 |
| Aug. | 125.7 | 69.1 | 141.1 |
| Sep. | 85.9 | 75.2 | · 111 •2 |
| Oct. | 47.6 | 62.8 | 72 •8 |
| Nov. | 23.7 | 41.2 | 40.5 |
| Dec. | 14 •4 | 22 •6 | 22 .2 |

Table (1)

This table is suited for all cases that under research. The collection efficiency is assumed to be 30%.

Case 1- Specific heat loss rate = 0.25 KW/oc.

Table 1.1 reveals the calculation of the effective solar contribution. The site under investigation has an annual number of 2800 degree-days or 67200 degreeh having, say, a 40 m² (for example) solar collector, then we have:

| | | THET | | | |
|-----------------|---|---|---|--|--|
| degc.h month | Space heat | Horiz. total | Unit collect- | Total collect- | Solar contrib. E but not |
| | requir. | radiat- | ion | ion ₂ | more than |
| | | ion | c x1 .5 x0 .3 | ;0 Dx+0 m [⊂] | В |
| | KWh | KWh/m ² | KWh/m ² | KWh | KWh |
| A | В | C | D | E | F |
| 10560 | 2640 | 18.3 | 8.235 | 329.4 | 329.4 |
| | 2400 | 30.9 | 13.905 | 556 •2 | 556.2 |
| - | 22 80 | 60.6 | 27.270 | 1090.8 | 1090.8 |
| | | 111.9 | 50.355 | 2014.2 | 1710.0 |
| | | | 55.440 | 2217.6 | 1182.0 |
| - | _ | | | | - |
| - | - | | 63.180 | | - |
| - | - | _ | | | - |
| 3096 | 774 | | 38.655 | | 774.0 |
| - | | | 21.420 | | 856.8 |
| | | - | | | 426.6 |
| 9840 | 2460 | 14.4 | 6.480 | 259.2 | 259.2 |
| 672 00 | 16800 | 933.0 | 419.850 | 16794.0 | 7185.0 |
| | A 10560 9600 9120 6840 4728 - 3096 5352 8064 9840 | month heat requir. 0.25xA KWh A B 10560 2640 9600 2400 9120 2280 6840 1710 4728 1182 | degc.h Space Horiz. month heat total requir. radiat- 0.25xA ion KWh KWh/m A B C 10560 2640 18.3 9600 2400 30.9 9120 2280 60.6 6840 1710 111.9 4728 1182 123.2 - - 150.4 - 140.4 - - 125.7 3096 3096 774 85.9 5352 1388 47.6 8064 2016 23.7 9840 2460 14.4 | degc.h Space Horiz. Unit month heat total collect- requir. radiat- ion cxl.5x0.3 KWh KWh/m² KWh/m² KWh/m² A B C D 10560 2640 18.3 8.235 9600 2400 30.9 13.905 9120 2280 60.6 27.270 6840 1710 111.9 50.355 4728 1182 123.2 55.440 - - 150.4 67.680 - - 140.4 63.180 - - 125.7 56.565 3096 774 85.9 38.655 5352 1388 47.6 21.420 8064 2016 23.7 10.665 9840 2460 14.4 6.480 | degc.h Space Horiz. Unit Total month heat total collect- collect- requir. radiat- ion ion 20100000000000000000000000000000000000 |

Table (1.1)

Table (1.2): Displays solar contribution and utilization rate for various collector sizes.

| All Values in KWh | Heating requir. B | Unit collection D | $\frac{10m^2}{total}$ D x 10 | net | 20m2tota1 D x 20 | net |
|----------------------|-------------------------|-------------------------|---------------------------------|------------|---------------------|---------|
| Jan. | 2 640 | 8.235 | 82 . 35 | 82.35 | 164 • 7 | 164.7 |
| Feb. | 2400 | 13.905 | 139.05 | 139.05 | 278.10 | 278.1 |
| Mar. | 2280 | 27.270 | 272.70 | 272.70 | 545.4 | 545.4 |
| Apr. | 1710 | 50.355 | 503.55 | 503.55 | 1007.1 | 1007.1 |
| May | 1182 | 55.440 | 554.40 | 554.40 | 1108.8 | 1108.8 |
| Jun. | - | 67.680 | 676.80 | - : | 1353.6 | - |
| Jul. | - | 63.180 | 631.80 | - 7 | 1163.6 | - |
| Aug. | - | 56.565 | 565.65 | - | 1131.3 | - |
| Sep. | 774 | 38.655 | 386.55 | 386.55 | | 773.10 |
| Oct. | 1 3 8 8 | 21.420 | 214.20 | 214.20 | | 428.40 |
| Nov. | 2016 | 10.665 | 106.65 | 106.65 | | 213.30 |
| Dec. | 2460 | 6.470 | 64.80 | 64.80 | 129.60 | 129.60 |
| lota1 | 16800 | 419.850 | 4198.50 | 2 324 .25 | 8397.00 | 4648.50 |
| Utiliz.rate |) | | | 0.5536 | | 0.5536 |

E.70. Mansoura Bulletin December 1978.

| | | | | | | <u> </u> |
|---------|----------------------------------|--------|-------------------|--------|---|----------|
| Month | 40 m² total | net | $60 m^2$ total | net | $\begin{array}{c} 80 \text{ m}^2 \\ \texttt{total} \end{array}$ | net |
| Jan. | 329.4 | 329.4 | 494•1 | 494•1 | 658.8 | 658.8 |
| Feb. | 556.2 | 556 •2 | 834.3 | 834.3 | 1112.4 | 1112.4 |
| Mar. | 1090.8 | 1090.8 | 1636.2 | 1636.2 | 2181.6 | 2181.6 |
| Apr. | 2014.2 | 1710.0 | 3021.3 | 1710.0 | 4028.4 | 1710.0 |
| Мау | 2217.6 | 1182.0 | 3326.4 | 1182.0 | 4435.2 | 1182.0 |
| June. | 2707.2 | - | 4060.8 | - | 5414.2 | - |
| Jul. | 2 52 7 •2 | - | 3790.8 | - | 5054 •4 | - |
| Aug. | 2262.6 | - | 3393•9 | - | 4525.2 | - |
| Sep. | 1546.2 | 774.0 | 2319.3 | 774•0 | 3092 .4 | 774 •0 |
| Oct. | 856.8 | 856.8 | 12 86 . 2 | 1286.2 | 1713.6 | 1388.0 |
| Nov. | 426.6 | 426.6 | 639.9 | 639.9 | 1253.2 | 1253-2 |
| Dec. | 259.2 | 259.2 | 388.8 | 388.8 | 518.4 | 518.4 |
| Total | 16794 | 7185.0 | 25191.06 | 8945.5 | 33588 | 10778.4 |
| Utiliz. | rate | 0.4278 | | 0.3551 | | 0.3209 |

Table (1.2) (Continue)

Table (1.3) reveals the costs of the auxiliary heating system needed at each collector size and the corresponding total costs for the two different capital costs of the solar collector (\pounds 15/m² and \pounds 10/m²) using various types of fuels.

Table (1.3): (Expected life time of the auxiliary heating system is 5 years). (a) Gas (£ 0.0073KWh x 5 = 0.0365).

| o <u>)l</u> ector Brea | Auxiliary heat per year | Costs of auxil. heat = £5x0.0073 | Cap.com (£ 10/1 | 2) cost ((<u>£10</u>) | C.cos (£15/m | |
|---------------------------|-------------------------------|---|--------------------|-----------------------------|-----------------|-----------------------|
| <u>n</u> ² | K Wh | KWh | | <u>л</u> 2 | | n ² |
| 0 | 16800.00 | 613.20 | 0 | 613.20 | 0 | 613.20 |
| 10 | 14475.75* | 528.36 | 100 | 628.36 | 150 | 679.36 |
| 20 | 12151.50 | 443.53 | 200 | 643.53 | 300 | 743.53 |
| 40 | 9615.00 | 350.95 | 400 | 750.95 | 600 | 950.95 |
| 60 | 7854.50 | 386.69 | 600 | 886.69 | 900 | 1186.69 |
| 80 | 6021.60 | 219.79 | 800 | 1019.79 | 12 00 | 1419.79 |

(heat requirement(16800))-(net solar contribution at 10 m2 (2324.25))=14475.75 KWh.

| | | | | • | | |
|----|------------|------------|---------|------------|--------------|---------|
| 0 | 16800.0 | 772.80 | 0 | 772.8 | 0 | 772.8 |
| 10 | 14475.75 | 665.88 | 100 | 765.88 | 150 | 815.88 |
| 20 | 12151.5 | 558.97 | 200 | 758.97 | 300 | 858.97 |
| 40 | 9615.0 | 442 •2 9 | 400 | 842 .2 9 | 600 | 1042.29 |
| 60 | 7854•5 | 361.31 | 600 | 961.32 | 900 | 1261.31 |
| 80 | 6021.6 | 276.99 | 800 | 1076.99 | 1200 | 1476.99 |
| | | | | | | |
| | | | | | | |
| | | | | | | |
| | (c) Fuel | 011 (£ 0.0 | 068/KWh | x 5 = 0.0 | 340) | |
| | 16800.0 57 | 1.2 0 | 571.2 | 2 0 | | 571 2 |

(b) Paraffin oil (£ 0.0092 KWh x 5 = 0.046)

| 0 | 16800.0 | 571 •2 | 0 | 571 •2 | 0 | 571 •2 |
|----|----------|--------|------|----------|------|---------|
| 10 | 14475.75 | 492.18 | 100 | 592 • 18 | 150 | 642.18 |
| 20 | 12151•5 | 413.15 | 2 00 | 613.15 | 300 | 713.15 |
| 40 | 9615.0 | 326.91 | 400 | 726.91 | 600 | 926.91 |
| 60 | 7854•5 | 267.05 | 600 | 867.05 | 900 | 1167.05 |
| 80 | 6021.6 | 204.73 | 800 | 1004.73 | 1200 | 1404.73 |

(d) House coal (£ 0.0051/KWh x 5 = 0.0255)

| 0 | 16800.0 | 428.4 | 0 | 428.4 | 0 | 426.4 |
|----|-----------------|---------|-----|----------|------|---------|
| 10 | 14475.75 | 369.13 | 100 | 469.13 | 150 | 519.13 |
| 20 | 12151.5 | 309.86 | 200 | 509.86 | 300 | 609.86 |
| 40 | 9615.0 | 245.18 | 400 | 645.18 | 600 | 845.18 |
| 60 | 7854.5 | 200.29 | 600 | 800 •2 9 | 900 | 1100.29 |
| 80 | 6 021. 6 | 55• ز15 | 800 | 953•55 | 1200 | 1353.55 |

****** Costs of auxiliary heat along 5 years using house coal = $16800 \times 0.0255 = \pounds 428.4$.

(e) Anthracite (£ $0.0049/KWh \times 5 = 0.0245$)

| 0 | 16800.0 | 411.6 | 0 | 411.6 | 0 | 411.6 |
|----|---------|---------|-----|-----------|------|----------|
| 10 | 14475.0 | 354.66 | 100 | 454.66*** | 150 | 504.66 |
| 20 | 12151.5 | 297.71 | 200 | 497.71 | 300 | 597.71 |
| 40 | 9615.0 | 235.57 | 400 | 635.57 | 600 | 835.57 |
| 60 | 7854.5 | 192 •44 | 600 | 792 •44 | 900 | 1092 •44 |
| 80 | 6021.6 | 147.53 | 800 | 947.53 | 1200 | 1347•53 |

******* Total costs along the 5 years of the heating system using $\pounds 10/m^2$ solar collector = 354.66 + 100 = $\pounds 454.66$.

E.72. Mansoura Bulletin December 1978.

| 0 | 16800.0 | 478.8 | 0 | 478.8 | 0 | 478 .8 |
|----|----------|--------|-----|----------------|-------------|------------------------|
| 10 | 14475.75 | 412.56 | 100 | 512 .56 | 150 | 562 . 56 |
| 20 | 12151.5 | 346.32 | 200 | 546•3 2 | 300 | 6 46•3 2 |
| 40 | 9615.0 | 274.03 | 400 | 674.03 | 600 | 874.03 |
| 60 | 7854.5 | 223.85 | 600 | 823.85 | 9 00 | 1123.85 |
| 80 | 6021.6 | 171.62 | 800 | 971.62 | 12 00 | 1371.62 |

(f) Coke (£ 0.0057/Kwh x 5 = 0.0285)

The computations are repeated and executed for 10 years and 15 years as life times of the auxiliary heating system.

All prevous results are also deduced when the specific heat loss rate = $0.5 \& 1.0 \text{ KW/}^{\circ}C$ at the prementioned constraints of different solar collector areas, various types of fuel & capital costs and 5, 10 & 15 years life times.

2.5. Conclusion and Comments on the results.

The expression of total costs is used for the costs of solar collector (copital cost) plus the cost of the auxiliary heating system used to withstand the heat requirement along the expected life-time of the latter. The total costs for various solar collector areas (0, 10, 20, 40, 60 and 80 m^2) are plotted as revealed by figures 1, 2,..., 9.

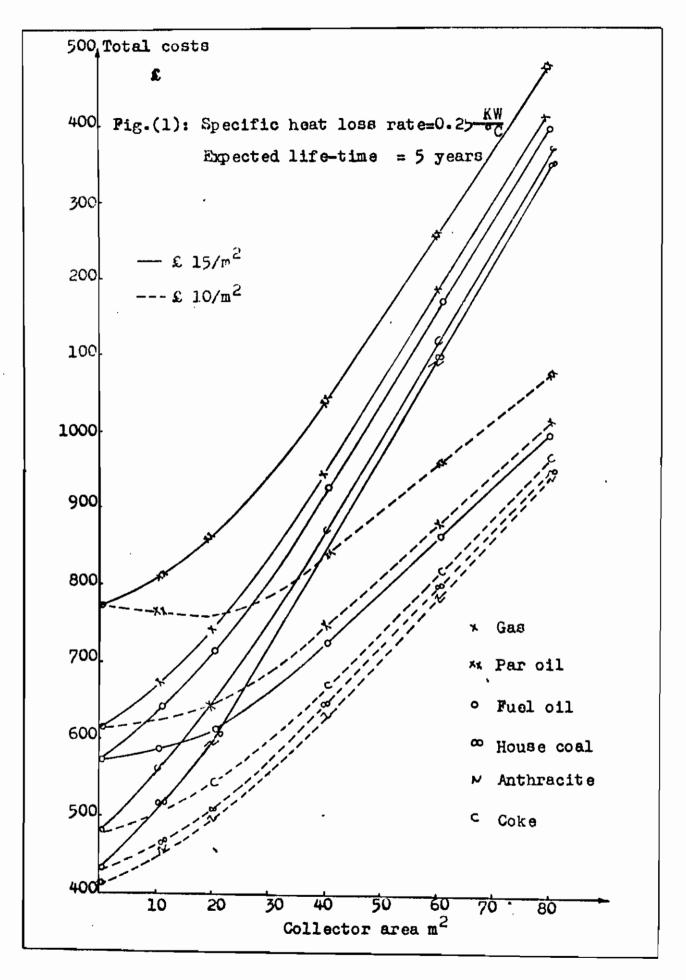
Figures 1, 2 & 3 displays case 1 when the specific heat loss rate = 0.25 KW/°C, where figures 4, 5 & 6 belong the second case of 0.5 KW/°C and finally case 3 is explained by the figures 7, 8 & 9.

Case 1. Specific heat loss rate = 0.25 KW/*C.

Family of curves are drawn for each expected life-time (5, 10 & 15 years), when the capital costs has the values of $\pounds 10/\text{m}^2$ (dotted lines) and $\pounds 15/\text{m}^2$ (solid lines) and for various types of fuel.

From figure 1, we remark that:

a) The variation of the total costs against solar collector area shows that they increase with a rapid rate.



- b) Paraffin oil at $\pounds 15/m^2$ leads to highest total costs compared with other types of fuel. However, anthracite at $\pounds 10/m^2$ yields the lowest total costs than various kinds of fuel taken.
- :) Using the solar collector under these conditions increases the total costs along the period taken hare (5-years).

Thus, we can satisfy with the auxiliary heating system only to fulfil the heat requirement.

Going now to figure 2 (10-years), one can conclude that:

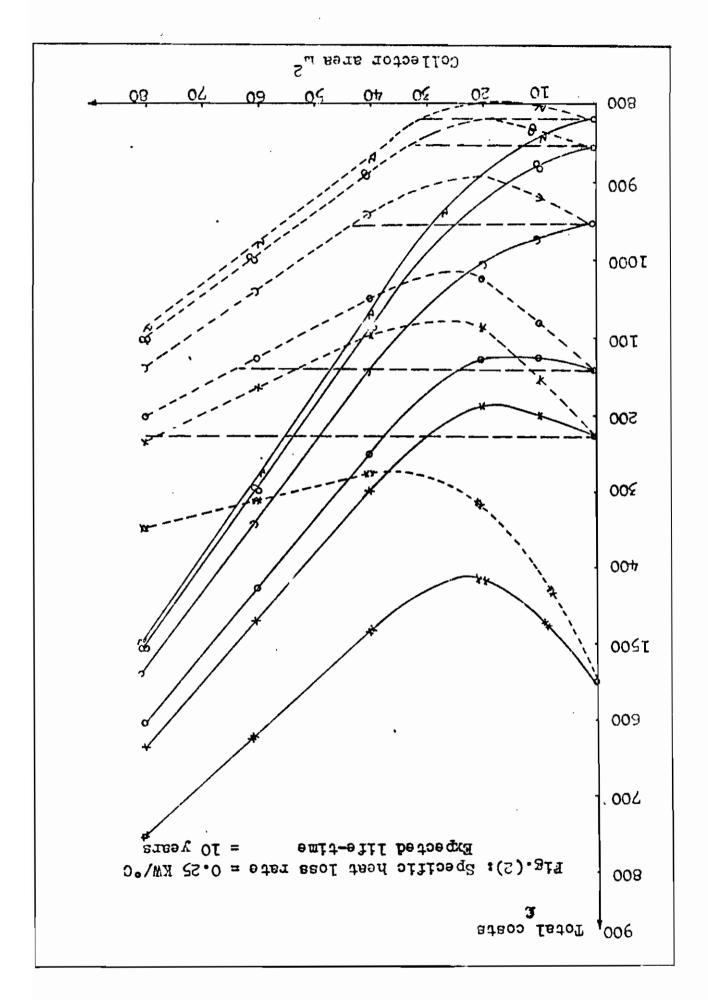
b) We have a minimum point (i.e. total cost) for all types of fuel taken but when the capital cost = £ 10/m². However, this point differs according to the type of fuel i.e. the optimum solar collector size is not the same as shown by the following table:

| | Expect a- Capita | = 10-years. = £ 10/m ² | |
|-----------------|---------------------------------|--------------------------------------|------------------------------|
| Type of fuel | Optimum collector size m2 | Corresponding total costs £ | Maximum collector size |
| Gas | 26 | 1080.00 | 78 |
| Parafin oil | 30 | 1275.00 | no max. |
| Fuel oil | 25 | 1015.00 | 64 |
| Ecuse coal | 20 | 819.70 | 32 |
| Anthracite | 20 | 795.42 | 30 |
| Coke | 20 | 892 .64 | 43 |

The optimum collector size is 20 m^2 The corresponding costs = £ 795.42

Specific heat loss rate = 0.25 KW/*C

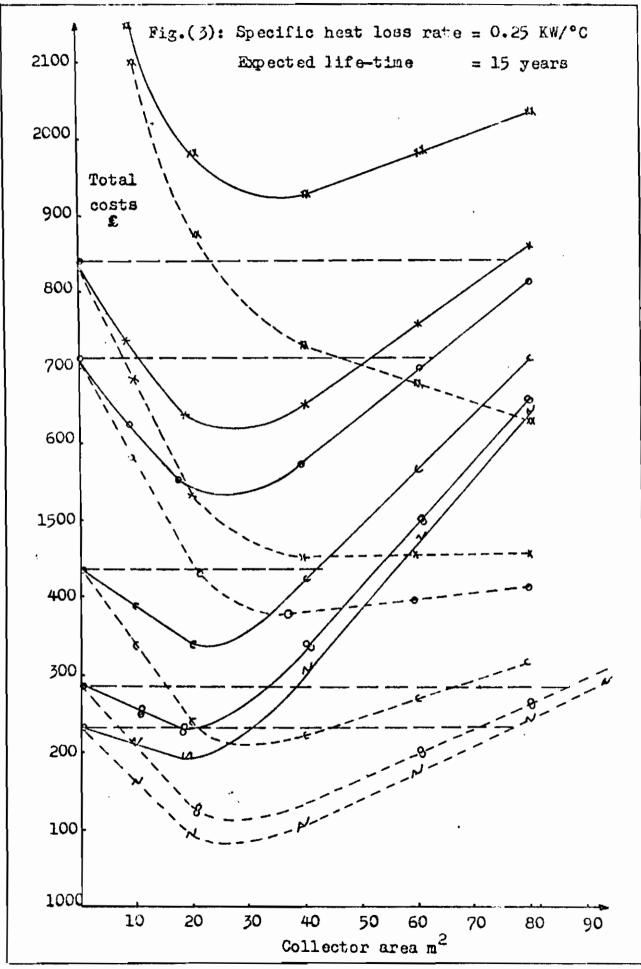
| | b- Capital cost | : | $= \pounds 15/m^2$ |
|--------------|------------------|---------|--------------------|
| Gas | 20 | 1187.06 | 30 |
| Paraffin oil | 21 | 1417.94 | 50 |
| Fuel oil | 20 | 1126.30 | 24 |
| House coal | no minimum point | | no max. |
| Anthracite | no minimum point | | 11 11 |
| Coke | no minimum point | | 11 11 |
| | | | |



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| 55 | <u>3</u> 225 | 20 | House coal |
| 62 | J 25 2 | 22 | [Ţo [əng |
| NO BEX. | 3 57 2 | 55 | Paraffin oil |
| 92 | ⊆ ⊺9⊺ | 56 | SED |

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The optimum collector size is 20 m^2 The corresponding costs £ 1190

Case 2. Specific heat loss rate = 0.5 KW/°C.

When we study figure 4. (Expected life-time is taken 5-years), then we can conclude that:

b) Paraffin oil has greater total costs for all collector sizes than other types. This occurs when the capital cost has f. 15/m² of solar collector, although anthracite has the lowest characteristic.

: •

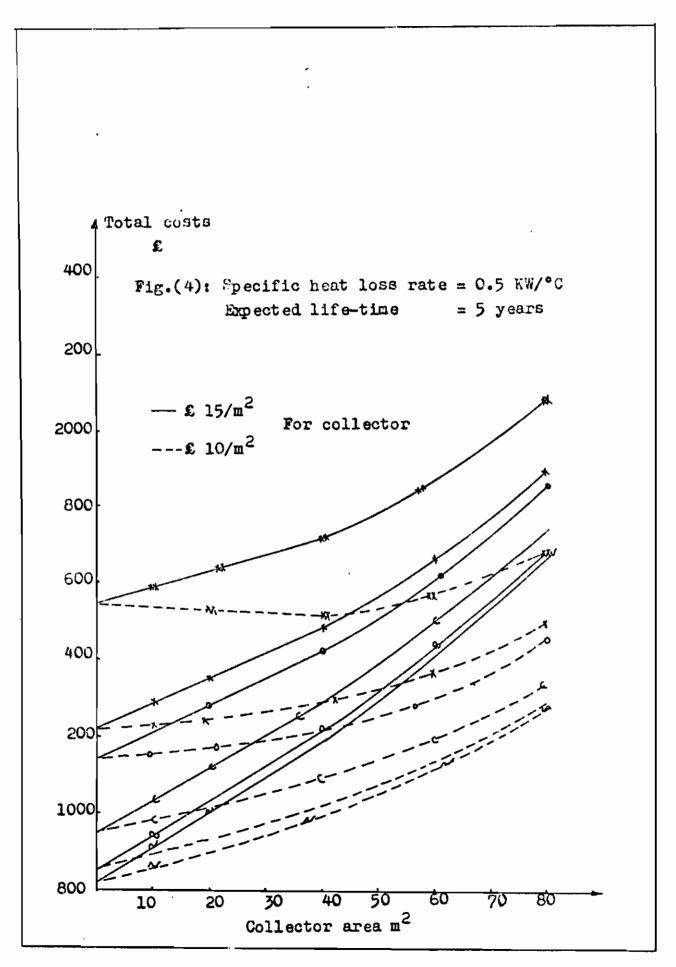
b) All the curves for different parameters have increasingly characteristics sxcept for paraffin oil at $\pounds 10/m^2$ which has a minimum point of 40 m² as a solar collector size with corresponding costs of $\pounds 1510$ total costs.

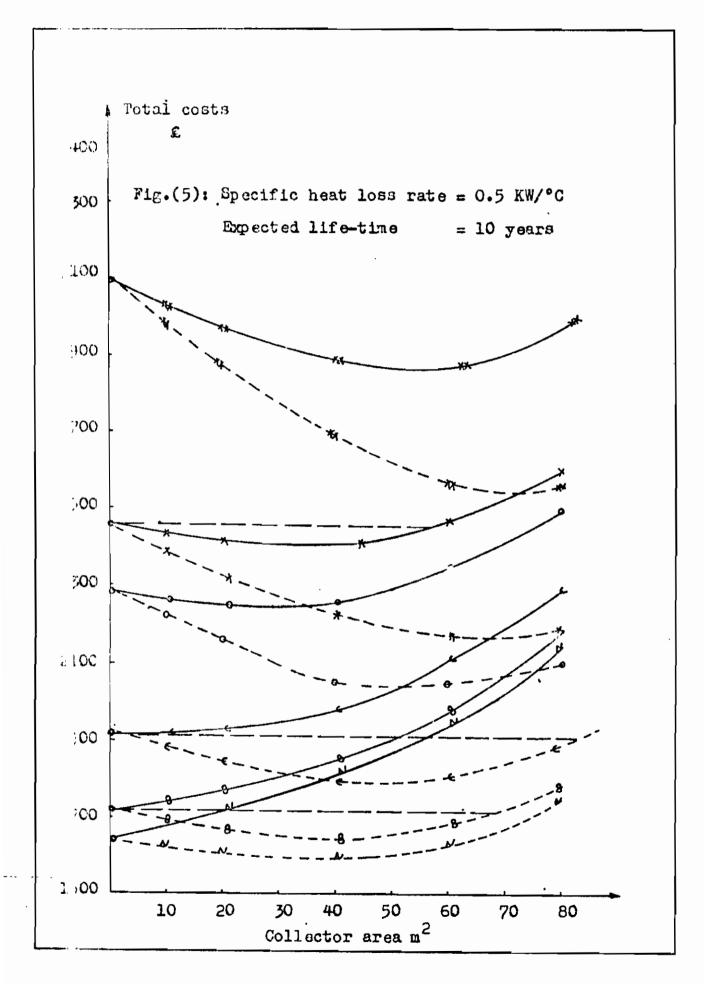
Investigating Fig. 5 (10 years), the optimum collector size for various parameters are tabulated as follows:

| | | / _ | |
|-----------------|---|-----------------------------------|---|
| Type of fuel | Optimum collector size m ² | Corresponding total costs £ | Maximum collector size m ² |
| Gas | 60 | 2160 | no max. |
| Par. oil | 71 | 2 54 0 | no max. |
| Fuel oil | 50 | 2040 | no max. |
| House coal | 40 | 1635 | 66 |
| Anthracite | 40 | 1585 | 60 |
| Coke | 43 | 1785 | 82 |
| | The optimum colle The corresponding b- Capital cost = £ | total costs is | |
| Ga s | 34 | 2400 | 58 |
| Par. oil | 55 | 2860 | 98 |
| Fuel oil | 33 | 2 240 | 47 |
| House coal | no minimum point | | no max. |
| Anthracite | no minimum point | 6 | 11 |
| Coke | no minimum point | ; | n |
| | | | |

a- Capital cost $\pounds 10/m^2$

The optimum collector size = 33 m^2 The corresponding total costs = £ 2240





э.

On studying Fig. 6(15 years), we summarize the optimum collector size and the maximum collector size beyond which the total costs is greater than that of the conventional one:

| Type of fuel | Optimum collector size m ² | Corresponding total costs £ | Maximum collector size m ² |
|-----------------|---|-----------------------------------|---|
| Gas | no minimum point | | no max. |
| Par. oil | 60 | 3885 | no max. |
| Fuel oil | no minimum point | | no max. |
| House coal | 50 | 2210 | no max. |
| Anthracite | 53 | 2160 | 113 |
| Coke | 60 | 2410 | no max. |

a- Capital cost = $\pounds 10/m^2$

The optimum collector size is 53 m²

| Gas | 49 | 3225 | no max. |
|------------|----|------|---------|
| Par. oil | 60 | 3885 | no max. |
| Fuel oil | 47 | 3040 | no max. |
| House coal | 40 | 2460 | 65 |
| Anthracite | 40 | 2380 | 60 |
| Coke | 40 | 2675 | 83 |

b- Capital cost = $\pounds 15/m^2$

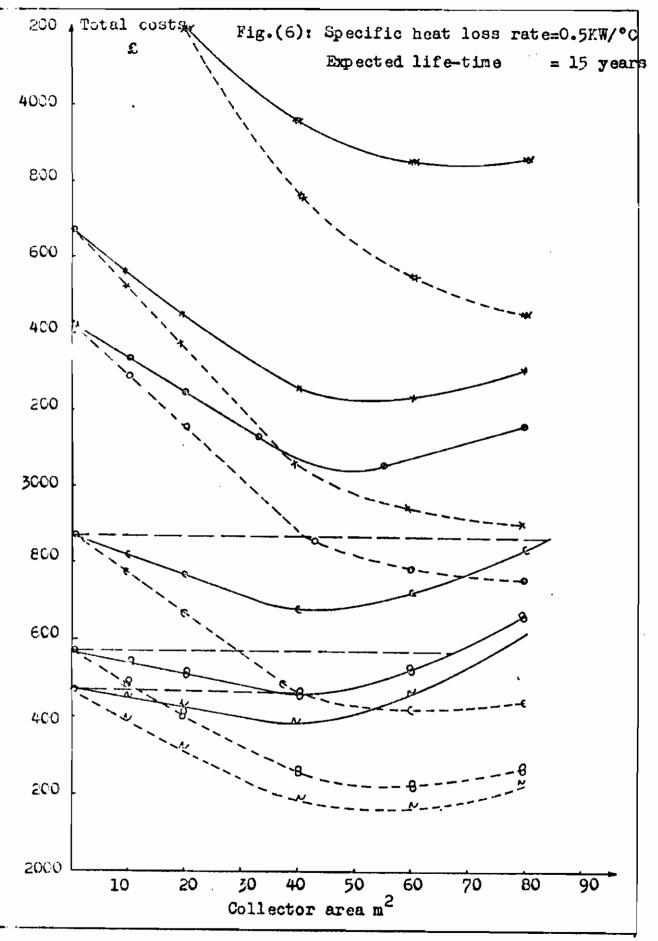
The optimum collector size is 40 m^2 The corresponding total costs = £ 2380

Case 3: Specific heat loss rate = 1.0 KW/°C

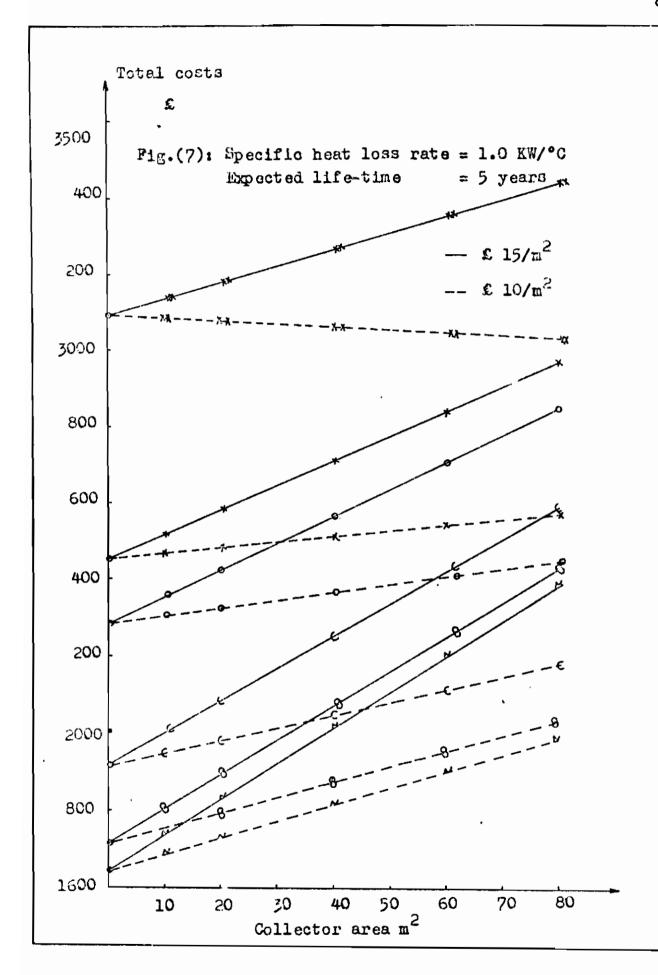
With respect to Fig. 7(5 years), we remark that all the characteristics are linearly varied and have no minimum point as in the preceding two cases. Moreover, these curves are increasingly functions i.e. on using larger size of solar collector, the total costs increase, except for paraffin oil (when capital cost = £ $10/m^2$), it has decreasingly function.

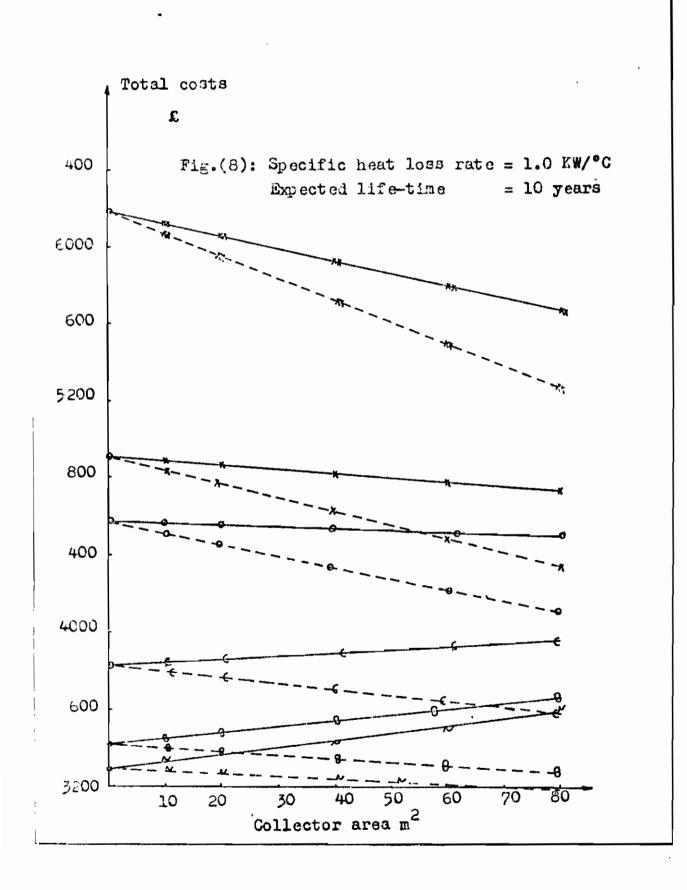
Researching for Fig. 8(10 years) we notice that the variations are linear but with increasingly, decreasingly and constant functions as shown.





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M.Helmy El-Maghraby

Paraffin oil is the more expensive type of fuel compared with other types under study, however, anthracite is the cheapest.

The fuel oil has a nearly constant characteristic i.e. it has the same total costs on using different sizes of solar collector.

Eventually, from Fig. 9, we remark that the corresponding family curves vary linearly against various collector sizes but with reducingly characteristics i.e. the total costs decrease on using larger solar collector size in the range chosen for the collector areas. This phenomenon occurs for all types of fuel.

2.6. Figure of merit (FM)

a_ Gas

It is defined as $\frac{\text{Energy saved by the install. in 10 years}}{\text{Extra cost of install. over conven one}}$

It has been suggested that the system will be competitive i (FM) reaches a value of 1.

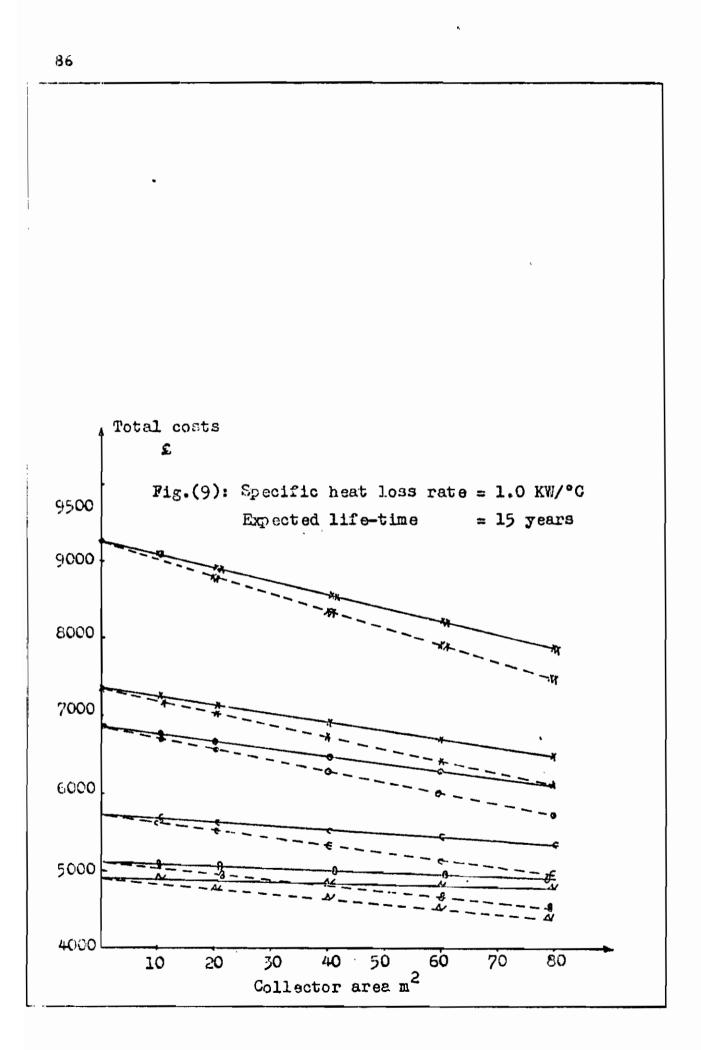
| Collector size m ² | Net solar contribut | Energy saved | Extra costs £10/m ² | FM (£10/m ²) | Extra costs £15/m ² | FM (£15/m ²) |
|-------------------------------------|------------------------|-----------------|--------------------------------------|-----------------------------|--------------------------------------|-----------------------------|
| m | (KWh)G | Gx0.073=H | | H/I | LI 57m I' | ш ² н/1́ |
| 10 | 2 324 •2 5 | 169.67 | 100 | 1.69 | 150 | 1.13 |
| 20 | 4648.50 | 339•34 | 200 | 1.69 | 300 | 1.13 |
| 40 | 7185.00 | 524 • 51 | 400 | 1.31` | 600 | 0.87 |
| 60 | 8945.50 | 653.02 | 600 | 1.09 | 900 | 0.73 |
| 80 | 10778.40 | 786 .8 2 | 800 | 0.98 | 12 00 | 0.66 |

Cose 1. Specific heat loss rate = 0.25 KW/C

Davaddin 12

| - | (£ 0.0092/KW | h x | 10 | years | = | £ | 0.092/KWh) |
|-----|--------------|-----|----|-------|---|---|------------|
| b++ | Parallin oll | | | | | | |

| 10 | 2 5• +2ز 2 | 213.83 | 100 | 2.14 | 1 50 | 1.43 |
|----|------------|----------|-----|------|------|------|
| 20 | 4648.50 | 427.66 | 200 | 2.14 | 300 | 1.43 |
| 40 | 7185.00 | 661.02 | 400 | 1.65 | 600 | 1.10 |
| 60 | 8945.50 | 822 • 99 | 600 | 1•37 | 900 | 0.91 |
| 80 | 10778.40 | 991.61 | 800 | 1.24 | 1200 | 0.83 |



c- Fuel oil
(£ 0.0068/KWh x 10 years = £ 0.068/KWh).

| | | | • | | | |
|----|------------|--------|-----|------|------|------|
| 10 | 2 324 .2 5 | 158.05 | 100 | 1.58 | 150 | 1.05 |
| 20 | 4648.50 | 316.09 | 200 | 1.58 | 300 | 1.05 |
| 40 | 7185.00 | 488.58 | 400 | 1.22 | 600 | 0.81 |
| 60 | 8945.50 | 608.29 | 600 | 1.01 | 900 | 0.68 |
| 80 | 10778.40 | 732.93 | 800 | 0.92 | 1200 | 0.61 |

d- House coal

 $(\pounds 0.0051/KWh \times 10 \text{ years} = \pounds 0.051/KWh).$

| 10 | 2 324 •2 5 | 118.54 | 100 | 1.19 | 150 | 0.79 |
|----|----------------------|--------|-----|------|------|------|
| 20 | 4648.50 | 237.07 | 200 | 1.19 | 300 | 0.79 |
| 40 | 7185.00 | 366.44 | 400 | 0.92 | 600 | 0.61 |
| 60 | 8 94 5.50 | 456.22 | 600 | 0.76 | 900 | 0.51 |
| 80 | 10778.40 | 549.69 | 800 | 0.69 | 1200 | 0.46 |

e- Anthracite

 $(£ 0.0049/KWh \times 10 \text{ years} = £ 0.049/KWh)$

| 10 | 2324 .25 | 213.89 | 100 | 1.14 | 150 | 0.76 |
|----|----------|---------|-----|------|------|------|
| 20 | 4648.50 | 227.78 | 200 | 1.14 | 300 | 0.76 |
| 40 | 7185.00 | 352 .07 | 400 | 0.88 | 600 | 0.59 |
| 60 | 8945.50 | 438.33 | 600 | 0.73 | 900 | 0.49 |
| 80 | 10778.40 | 528.14 | 800 | 0.66 | 1200 | 0.44 |

f- Coke

| $(\pounds 0.0057/KWh \times 10 \text{ years} = \pounds 0.057/KWh$ |
|---|
|---|

| 10 | 2 324 .25 | 132.48 | 100 | 1 • 32 | 150 | 0.88 |
|----|------------------|----------|-----|--------|------|------|
| 20 | 4648.50 | 264.96 | 200 | 1 • 32 | 300 | 0.88 |
| 40 | 7185 . 00 | 409.55 | 400 | 1.02 | 600 | 0.68 |
| 60 | 2945.50 | 509.89 | 600 | 0.85 | 900 | 0.57 |
| 80 | 10778.40 | 614 • 37 | 800 | 0.77 | 1200 | 0.51 |
| | | | | | | |

a- Gas

| Collector size m ² | Net solar contribut | Energy saved Gr0.073=1 | Extra costs (510/m2) | FM (£10/m ²) | Extra costs (£15/m²) | FM (£15/m ²) |
|-------------------------------------|------------------------|------------------------------|----------------------------|-----------------------------|----------------------------|-----------------------------|
| | G | | I | H/I | I´ | R/I |
| 10 | 2 324 •25 | 169.67 | 100 | 1.69 | 150 | 1.13 |
| 20 | 4648.50 | 339.34 | 200 | 1.69 | 300 | 1.13 |
| 40 | 92 97 • 00 | 678.68 | 400 | 1.69 | 600 | 1.13 |
| 60 | 12212.80 | 891 • 53 | 600 | 1.49 | 900 | 0.99 |
| 80 | 14370.00 | 1049.01 | 800 | 1.31 | 1200 | 0.87 |

Case 2. Specific heat loss rate = 0.5 KW/°C

b- Paraffin oil

(£ 0.0092/KWh x 10 years = £ 0.092/KWh)

| 10 | 2324 • 25 | 213.83 | 100 | 2.14 | 150 | 1.43 |
|----|------------|---------|-----|--------|-------|------|
| 20 | 4648.50 | 427.66 | 200 | 2 • 14 | 300 | 1•43 |
| 40 | 92 97 • 00 | 855.32 | 400 | 2•14 | 600 | 1.43 |
| 60 | 12212.80 | 1123.58 | 600 | 1.87 | 900 | 1.25 |
| 80 | 14370.00 | 1322.04 | 800 | 1.65 | 12 00 | 1.10 |

c- Fuel oil

(£ 0.0068/KWh x 10 years = £ 0.068/KWh)

•

| | • | • | • | | | |
|----|------------|--------|-----|------|-------|------|
| 10 | 2324 • 25 | 158.05 | 100 | 1.58 | 150 | 1.05 |
| 20 | 4648.50 | 316.09 | 200 | 1.58 | 300 | 1.05 |
| 40 | 92 97 • 00 | 632.19 | 400 | 1.58 | 600 | 1.05 |
| 60 | 12212.80 | 830.47 | 600 | 1.38 | 900 | 0.92 |
| 80 | 14370.00 | 977.16 | 800 | 1.22 | 12 00 | 0.81 |

d- House coal (£ 0.0051/KWh x 10 years = £ 0.051/KWh)

| 10 | 2324.25 | 118.54 | 100 | 1.19 | 150 | 0.79 |
|----|------------|------------------|-----|------|-------|------|
| 20 | 4648.50 | 237.07 | 200 | 1.19 | 300 | 0.79 |
| 40 | 92 97 • 00 | 474.15 | 400 | 1.19 | 600 | 0.79 |
| 60 | 12212.80 | 62 2 . 85 | 600 | 1.04 | 900 | 0.69 |
| 80 | 14370.00 | 732.87 | 800 | 0.92 | 12 00 | 0.61 |

e- Anthracite

 $(£ 0.0049/KWh \times 10 \text{ years} = £ 0.049/KWh)$

| 10 | 2 324 • 25 | 113.89 | 100 | 1.14 | 150 | 0.76 |
|----|------------|----------|-----|------|------|------|
| 20 | 4648.50 | 227.78 | 200 | 1.14 | 300 | 0.76 |
| 40 | 9297.00 | 455.55 | 400 | 1.14 | 600 | 0.76 |
| 60 | 12212.80 | 598.43 | 600 | 0.99 | 900 | 0.66 |
| 80 | 14370.00 | 704 • 13 | 800 | 0.88 | 1200 | 0.59 |
| 80 | 14370.00 | 704.13 | 800 | 0.88 | 1200 | 0. |

f- Coke

_

| | (£ 0.005 | 57/KWh x 1 | 0 years | = £ 0.05 | 7/KWh) | |
|----|----------|------------|---------|----------|--------|------|
| 10 | 2324 •25 | 132.48 | 100 | 1•32 | 150 | 0.88 |
| 20 | 4648.50 | 264.96 | 200 | 1.32 | 300 | 0.88 |
| 40 | 9297.00 | 529.93 | 400 | 1.32 | 600 | 0.88 |
| 60 | 12212.80 | 696.13 | 600 | 1.16 | 900 | 0.77 |
| 80 | 14370.00 | 819.09 | 800 | 1.02 | 12 00 | 0.68 |

Case 3. Specific heat loss rate = 1.0 KW/°C

| | a- Gas (£ 0.00 | 73/KWh x 1 | O years : | =£0.07 | 3/KWh) | |
|----|-----------------------|------------------|-----------|--------|--------|------|
| 10 | 2324 •25 | 169.67 | 100 | 1.69 | 150 | 1.13 |
| 20 | 4648.50 | 339.34 | 2 00 | 1.69 | 300 | 1.13 |
| 40 | 92 97 • 00 | 678.68 | 400 | 1.69 | 600 | 1.13 |
| 60 | 13945.50 | 1018.02 | 600 | 1.69 | 900 | 1•13 |
| 80 | 185 94 •00 | 1357 .3 6 | 800 | 1.69 | 1200 | 1.13 |

b- Paraffin oil (£ 0.0092/KWh x 10 years = £ 0.092/KWh)

| 20 40 | 4648.50 9297.00 | 427.66 855.32 | 2 00 4 00 | 2•14 2•14 | 300 600 | 1•43 1•43 |
|----------|-----------------------|------------------|--------------|--------------|------------|--------------|
| 60 | 13945.50 | 1282.99 | 600 | 2.14 | 900 | 1.43 |
| 80 | 185 94 .00 | 1710.65 | 800 | 2.14 | 12 00 | 1.43 |

c- Fuel oil

 $(\pounds 0.0068/KWh \times 10 \text{ years} = \pounds 0.068/KWh)$

| 10 | 2 32 4 • 2 5 | 158.05 | 100 | 1.58 | 150 | 1.05 |
|----|-------------------|----------|------|------|------|------|
| 20 | 4648.50 | 316.09 | 2 00 | 1.58 | 300 | 1.05 |
| 40 | 92 97 .0 0 | 632 • 19 | 400 | 1.58 | 600 | 1.05 |
| 60 | 13945.50 | 948.29 | 600 | 1.58 | 900 | 1.05 |
| 80 | 18594.00 | 1264.39 | 800 | 1.58 | 1200 | 1.05 |

d- House coal

| | $(\pounds 0.0051/KWh \times 10 \text{ years} = \pounds 0.051/KWh)$ | | | | | | |
|----|--|---------|-----|------|------|------|--|
| 10 | 2 324 •2 5 | 118.54 | 100 | 1.19 | 150 | 0.79 | |
| 20 | 4648.50 | 237.07 | 200 | 1.19 | 300 | 0.79 | |
| 40 | 92 97 .00 | 474•15 | 400 | 1.19 | 600 | 0.79 | |
| 60 | 13945.50 | 711 •22 | 600 | 1.19 | 900 | 0.79 | |
| 80 | 18594.00 | 948.29 | 800 | 1.19 | 1200 | 0.79 | |

e- Anthracite (£ 0.0049/KWh = 10 years = £ 0.049/KWh)

| | | • | | | | |
|----|------------|--------|-----|------|-------|------|
| 10 | 2 324 •2 5 | 113.59 | 100 | 1•14 | 150 | 0.76 |
| 20 | 4648.50 | 227.78 | 200 | 1•14 | 300 | 0.76 |
| 40 | 9297.00 | 455.55 | 400 | 1.14 | 600 | 0.76 |
| 60 | 13945.50 | 683.33 | 600 | 1.14 | 900 | 0.76 |
| 80 | 18594.00 | 911.11 | 800 | 1•14 | 12 00 | 0.76 |

f- Coke (£ 0.0057/KWh x 10 years = £ 0.057/KWh)

| 10 | 2 324 +2 5 | 1 32 .48 | 100 | 1•32 | 150 | 0.88 |
|----|------------|----------|-----|--------|-------|------|
| 20 | 4648.50 | 264.96 | 200 | 1•32 | 300 | 0.88 |
| 40 | 92 97 .00 | 529.93 | 400 | 1 • 32 | 600 | 0.88 |
| 60 | 13945.50 | 794 •89 | 600 | 1•32 | 900 | 0.88 |
| 80 | 18594.00 | 1059.86 | 800 | 1.32 | 12 00 | 0.88 |
| | | | | | | |

2.7. Conclusions and comments on:

Utilization rate (UR) and figure of merit (FM) curves.

I) Utilization rate (UR)

UR expresses the effective solar collected energy utilized for satisfying all or part of the heat requirement as a ratio to the total collected energy. Fig. 10 displays the behaviour of UR against the solar collector areas at different values of specific heat loss rate. For small

As this rate increases (0.5 and 1.0 KW/°C), UR has a constant value at small collector sizes and then decreases at larger one.

When the specific heat loss rate = 1.0, UR possesses a constant value independent of solar collector areas.

values of this rate. UR decreases by a non-linear relation.

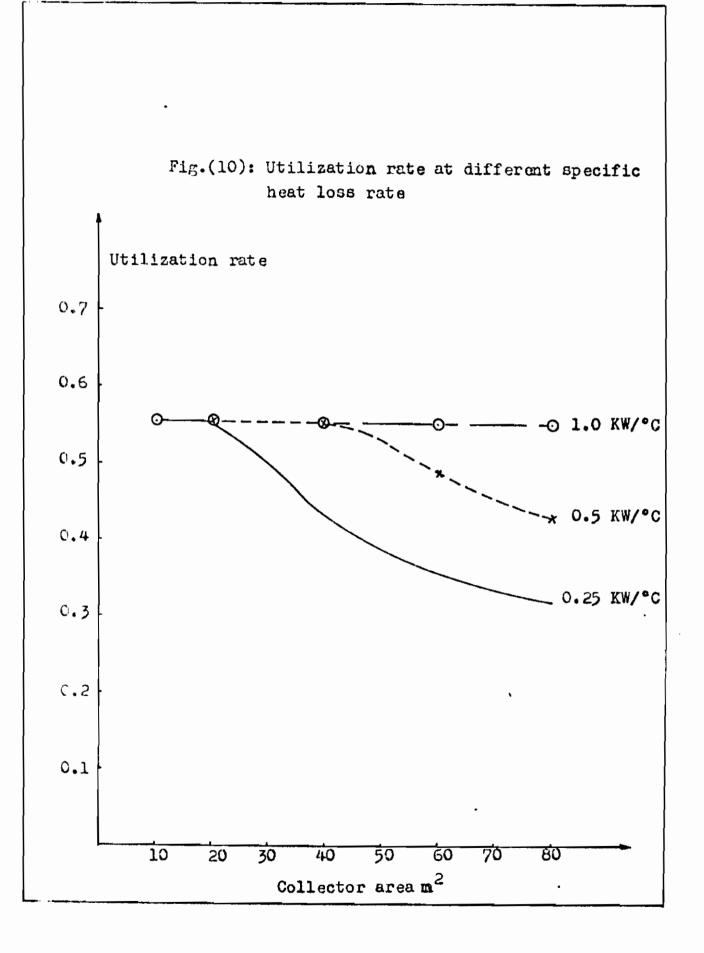
II) Figure of merit (FM)

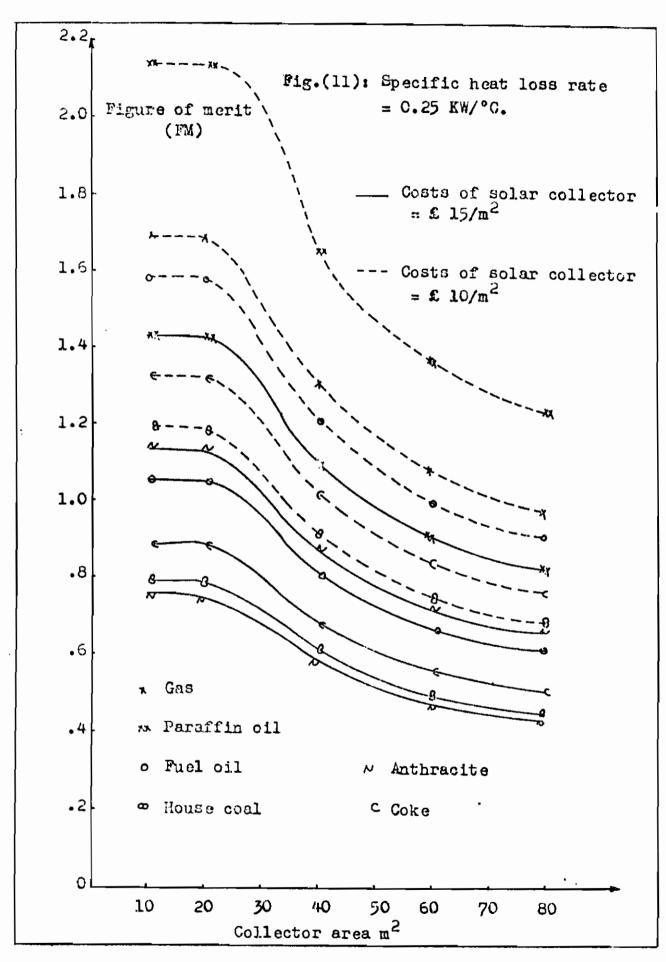
FM is plotted for 3 main cases of specific heat loss rate (0.25, (Fig. 11), 0.5(Fig.12) and 1.0(Fig. 13)).

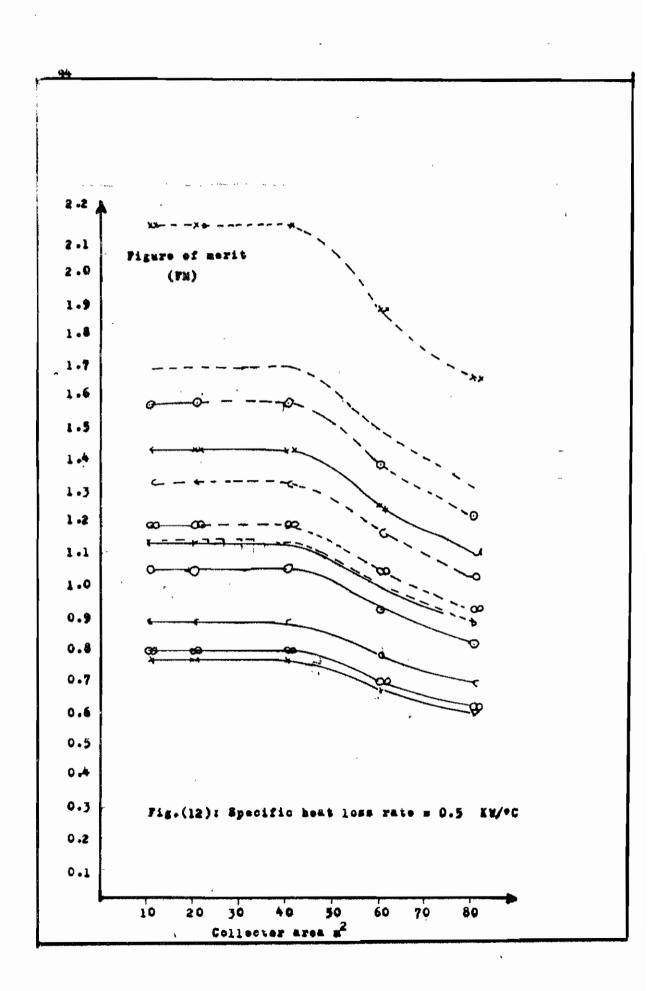
Fig. 11 shows the variations of FM against the solar collector area for different types of fuel. We notice that the paraffin oil has the greatest ratio, however, it has the same shape as other characteristics.

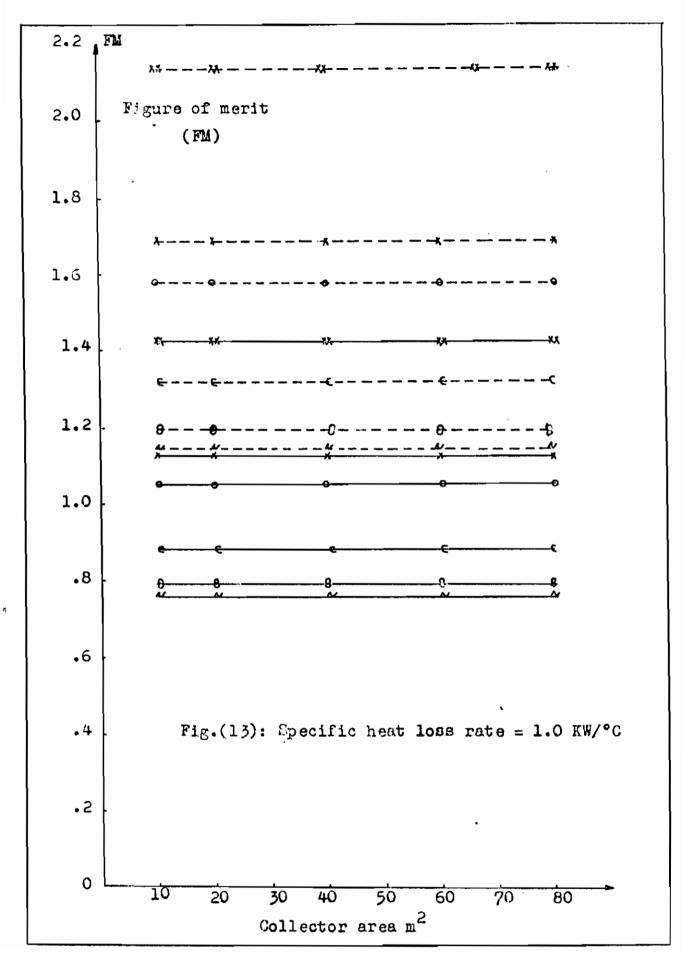
This means that maximum energy is saved along the period of research on using solar collector with £ $10/m^2$ as a capital cost and paraffin oil as a fuel.

On the other hand, anthracite yields minimum energy saved on using \pounds 15/m² as a capital cost.









9k M.Helmy El-Maghraby

Fig. 12 reveals the variation but with specific heat loss ate = 0.5 KW/°C. Also, Paraffin oil achieves the same FM of taximum energy saved relative to the extra cost of solar colletor installation. In addition, on using anthracite as a fuel, the can have minimum FM.

There is a difference between the characteristics here and inat in the case of 0.25 KW/°C. It is the constancy of FM(here) long 10, 20, 30 & 40 m² solar collector i.e. independent of sing larger sizes in this range, however, on increasing the plar collector area more than 40 m², FM decreases slowly with on-linear characteristic.

Fig. 13 explains the third case of specific heat loss rate 13 1.0 KW/°C. We notice a very significant phenomenon which is summarized as follows:

For all types of fuel, FM has a constant but with different we have independent of the size of the solar collector. Similarly, paraffin oil has a maximum FM on using $\pounds 10/m^2$ as a capital cost with respect to other types of fuel.

Again, anthracite yields minimum FM but it has a constant Wilke against the solar collector sizes.

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